# NUMERICAL SIMULATION OF STEAM AND WATER PRODUCING WELL **TESTING**

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### **ABSTRACT**

The main method of well and high temperature hydrothermal reservoirs testing at different stages of their exploration is production flow test of a well producing steam or steam and water mixture with different vapor saturation The data obtained are used to find the correlation between mass production rate and well head pressure that represents very important characteristics of well productive capacity. However the use of these data to obtain the characteristics of the productive zones and the processes generated in them is still not enough investigated problem. We have developed the methods of numerical simulation that makes it possible to take into consideration the different conditions of fluid flow in the productive zones and in the wellbore and thus to obtain the calculated relationships between mass production rate of the fluid as well as its enthalpy and the pressure at the well head. As the example driven shows the use of numerical simulation provides obtaining additional information about the process parameters that leads to higher effectiveness of exploration works.

## 1. INTRODUCTION

The main purpose of standard field investigations carried out at the testing of steam and water producing wells and high temperature hydrothermal reservoirs is to get the following data:

- pressure and enthalpy of steam and water mixture at the well outlet  $p_k$ ,
- mass production rate and regime of fluid discharge at different well head pressure;
- -mass concentration of water and steam in the fluid prod-
- -chemical and gaseous composition of water and steam.

Considering these data as a very significant information about the productive capacity of the well tested it should be noted that they do not provide obtaining the number of other parameters important for hydrothermal reservoir investigation (phase state of the fluid in the productive zone, its fluid conductivity, etc.). Attempts to use traditional methods of hydrodynamic investigations by plotting  $p - \ln \tau$  and  $p - \tau / r^2$  diagrams proved to be inefficient at high temperature hydrothermal reservoirs because of the effect of nonisothermal filtration and fluid phase state changing in the process of its filtration in the productive zone (Kiryukhin et al., 1991).

Nevertheless, the existing methods of well testing that provide the obtaining  $Q = f(p_h)$  and  $e = f(p_h)$  diagrams make it possible to identify the conditions of steam and water producing well testing in more detail and thus to increase the informativeness of testing results. The identification of the well testing conditions can be made on the base of numerical simulation combined with stochastic variation of the process parameters, which characterize the fluid flow conditions in the productive zone and in the wellbore. Results of the combined simulation compared with the observed data of the process development  $(Q = f(p_h))$  and  $e = f(p_h)$  diagrams) allow to determine with some degree of insured probability to what particular type of fluid flow conditions these data can be related. To realize this approach it is necessary to develop the mathematical model of well and productive zone compatible performance that can be used as the base for the numerical simulation of the well testing processes.

# 2. METHODS OF NUMERICAL SIMULATION OF WELL AND PRODUCTIVE ZONE COMPATIBLE PERFORMANCE

Fluid in the productive zone of the high temperature hydrothermal reservoir can be in one-phase (superheated water or steam) or in two-phase state (SWM) depending on the geothermal conditions and well depth. In the process of production flow testing initially onephase state of the fluid can be changed into two-phase one if the pressure in the wellbore where it intersects the productive zone drops to the saturation pressure corresponding to the temperature in that interval. Considering possible phase transfer of the fluid the general set of equations for the non-isothermal fluid filtration in the productive zone we represent by the equations of mass conservation

$$\partial(m\rho)/\partial\tau = q_m - \operatorname{div}(V' + V''), \qquad (1)$$

$$V'' = \log P'' \langle T_{11} = \sigma''_{12} \rangle / \sigma''_{13} \tag{2}$$

$$V'' = -k\rho''R''(\nabla p - \rho''g)/\mu'',$$
and energy conservation (3)

$$\partial E / \partial \tau = \nabla (\lambda \nabla t) - \operatorname{div} (V'e' + V''e'') + q_e, \qquad (4)$$

where 
$$\rho = \rho'(1-S) + \rho''S$$
  
 $E = mpe + (1-m)\rho_0 e_0;$   
 $e = \eta e'' + (1-\eta)e'$   
 $e_0 = c_0 t$   
 $\eta = V''/(V' + V'')$   
 $I = m[\lambda'(1-s) + \lambda''s] + (1-m)\lambda_0$   
(Piscacheva, 1991).

Set of equations (1) - (4) is related to non-isothermal one- or two-phase filtration in a porous or fractured medium where rock matrix and fluid are considered to be in local thermal equilibrium.

In response to the problems of well and productive m e compatibility simulation equations (1) - (4) were transferred to the cylindrical coordinates r, z with the substitution of new variable

$$x = r^{2} \text{ and represented as follows}$$

$$\beta^{*} \frac{\nabla r}{\partial \tau} = 4 \frac{\partial}{\partial x} \left( xT \frac{\partial p}{\partial x} \right) + \frac{\partial}{\partial x} \left( T \frac{\partial p}{\partial x} \right) + \frac{\partial}{\partial x} \left( T \frac{\partial p}{\partial x} \right) - g \frac{\partial}{\partial z} \left( T' \rho' + T'' \rho'' \right) + q_{m}, \qquad (5)$$

$$\alpha^{*} \frac{\partial e}{\partial x} = 4 \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x} \right) + \frac{\partial}{\partial x} \left( x\lambda \frac{\partial t}{\partial x}$$

$$+\frac{\partial}{\partial z}\left(\lambda\frac{\partial t}{\partial z}\right) - 4xV_x\frac{\partial e}{\partial x} - V_z\frac{\partial e}{\partial z} + q_e, \qquad (6)$$

where 
$$\beta^* = m\partial \rho / \partial p + (1-m)\rho\beta_3$$
  
 $\alpha^* = m\rho + (1-m)c_3\partial t / \partial e$ ;  
 $T' = k\rho'R' / \mu'$ ;  
 $T'' = k\rho''R'' / \mu''$   
 $T = T' + T'$ .

Set of equations (5) • (6) is solved numerically with the use of fully implicit finite difference method that approximates the original equations to the following schemes (Samarsky, 1977)

$$C\frac{y^{n+\frac{1}{2}}-y^n}{A\tau}=4\Lambda_i y^{n+\frac{1}{2}}+\varphi_l, \qquad (7)$$

$$C\frac{y^{n+1}-y^{n+\frac{1}{2}}}{A\tau} = A_{j}y^{n+1} + \varphi_{2}, \qquad (8)$$

where y is pressure or energy value, Ar is time step. Calculational ana is discretizied into N horizontal steps of h, and M vertical steps of  $h_1$  size. For the general case of non-uniform discretization the average steps will be

$$h_i = (h_i + h_{i+1})/2$$
,  $i = 1,2...(N-1)$ ,  
 $h_i = (h_i + h_{i+1})/2$ ,  $j = 1,2...(M-1)$ .

When the filtration equation is solved operator  $A_{\alpha}y^{t}$  is referred to

$$A_{\alpha}y^{t} = a_{\alpha+1} \frac{p_{\alpha+1}^{t} - p_{\alpha}^{t}}{h_{\alpha+1}} - a_{\alpha} \frac{p_{\alpha}^{t} - p_{\alpha-1}^{t}}{h_{\alpha}}, \ \alpha = i, j,$$
(9)
It is assumed that  $t = n + \frac{1}{2}$ , if  $\alpha = i$ , and  $t = n + 1$ , if  $\alpha = j$ . The

coefficients  $\varphi$ , C and  $a_{\alpha}$  are calculated as follows:

$$\begin{split} & \varphi_{l} = q_{m}/2 \; ; \qquad \varphi_{2} = q_{m}/2 - \\ & - g(a'_{l+1} \rho'_{l+1} + a''_{l+1} \rho''_{l+1} - a'_{l} \rho'_{l-1} - a'_{l} \rho''_{l-1})/\hbar_{j} \; ; \\ & C = \beta^{\hbar} \; ; \qquad a'_{l} = x_{l-1} \sum_{i=1}^{n} T'_{l} \; T'_{l-1} \ln(r_{l}/r_{l-1})/ \\ & / [T'_{l-1} \ln(r_{l}/r_{l-1}) + T'_{l} \ln(r_{l-1}/r_{l-1})]\hbar_{l} \; ; \\ & a'_{l} = 2T'_{l} \; T'_{l-1}/(T'_{l} + T'_{l-1})\hbar_{j} \; ; \qquad a_{\alpha} = a'_{\alpha} + a''_{\alpha} \; . \end{split}$$

Coefficients  $a'_{i+1}$ ,  $a'_{j+1}$ , etc. are determined the same way.

When the equation of energy is solved operator  $\Lambda_{\alpha} y^{I}$  is

$$\Lambda_{a}y' = r_{a}^{+} \frac{e'_{a+1} - e'_{a}}{h_{a+1}} + r_{a}^{-} \frac{e'_{a} - e'_{a-1}}{h_{a}}, a = i, j$$
 (10)  
The coefficients are determined by the following expressions:

$$\begin{aligned} \varphi_{1} &= q_{e} / 2 + \left[ x_{1+1/2} \lambda_{i+1/2} \left( t_{i+1} - t_{i} \right) / h_{i+1} \right. \\ &- x_{1-1/2} \lambda_{1-1/2} \left( t_{i} - t_{i-1} \right) / h_{11} / h_{i}; \\ \varphi_{2} &= q_{e} / 2 + \left[ \lambda_{j+1/2} \left( t_{j+1} - t_{j} \right) / h_{j+1} \right. \\ &- \lambda_{j-1/2} \left( t_{j} - t_{j-1} \right) / h_{j} \right] / h_{j}; \\ r_{i} &= - x_{i} V_{i} ; r_{j} = - V_{j} ; r_{\alpha}^{+} = \left( r_{\alpha} + \left| r_{\alpha} \right| \right) / 2 \\ r_{\alpha}^{-} &= \left( r_{\alpha} - \left| r_{\alpha} \right| \right) / 2. \end{aligned}$$

Values of y at i=1,N and j=1,M are to be found according to the given border conditions

Mathematical model of two-phase flow in a wellbore can be represented by the equations of motion

$$-\partial p/\partial Z = \psi \rho'' g + (I - \psi)\rho' g + 2\tau_k / r_w,$$
 and energy conservation (11)

$$\partial(\rho e)/\partial \tau = q_w - \partial(Ve)/\partial z + 2\tau_k v/r_w,$$
where  $v = \psi v' + (1 - \psi)v'$ 

$$V = \psi \rho'' v'' + (1 - \psi)\rho' v'$$
(12)

(Mamaev, et al, 1978, Kutateladze, 1973, Berglas and Sju, 1970, Aver/ev, 1960, Droznin, 1980, Zabarny, et al, 1992). The main difficulties in the solution of equations (11) - (12) are o with the variety of possible types of two-phase fluid flow in the wellbore influencing the hydraulic resistance to flow and heat exchange between two-phase flow and surrounding rocks.

Summarizing the results of previous studies of two-phase flow in the pipes and wellbores we have obtained the relations that provide the determination of the shear stress value  $\tau_k$  for different patterns of two-phase fluid flow in the wellbore (pellet or annular type) as well as the criteria for the type of flow pattern identification (Piscacheva, 1991). This made it possible to find the relationship between the pressure gradient in the wellbore and mass production

$$(\partial p/\partial z)_f = 2\tau_k/r_w = FQ^2, \qquad (13)$$

where the coefficientF is dependent on the pattern type of two-phase fluid flow. As far as the intensity of the heat exchange is concerned it is found that it can be calculated as follows

$$g_w = 2K_x[t(z) - t]/r_w$$
, (14)

where  $K_{\mathbf{r}}$  is transient heat exchange coefficient determined by the expression (Pudovkin, et al., 1977)

$$K_{\tau} = k_{t}/[1 + Bi \ln (1 + \sqrt{\sigma})], \qquad (15)$$

t(z) is initial temperature of rocks at the depth; t is temperature of the two-phase fluid at the same depth;

$$Bi = \alpha_0 r_w / \lambda_0$$

$$Fo = a_0 \tau / r_w^2$$

$$a_0 = \lambda_0 / (\rho_0 c_0)$$

 $\gamma$  " coefficient approximating the time dependence of  $K_{r}$  (if  $Bi < 30 \ \gamma = 2$ , if  $Bi > 30 \ \gamma = \pi$ );

$$k_t = 1/(1/\alpha_0 + \delta_c/\lambda_c + \delta_t/\lambda_t),$$

Set of equations (11) - (12) is solved by finite difference method using the explicit schemes. The equation of motion (11) is solved first accounting to the pressure value found at the wellbore from the solution of equations (5) - (6) for the fluid flow in the productive zone or to the accepted pressure at the well head The pressure distribution along the wellbore obtained is used then to solve the equation of energy (12). After that all coefficients in the equations (11) and (12) are recalculated according to the obtained values of pressure and enthalpy and set of equations (11) - (12) is solved again. Phase state of the fluid in the wellbore is determined by the value of its specific enthalpy. If  $e(z) > e'_{x}$  (  $e'_{x}$  is the specific enthalpy of saturated water) it means that at the depth z phase transfer takes place. In order to provide the compatible performance of a well and productive zone set of equations (5) - (6) and that of equations (11) - (12) must be simultaneously satisfied that requires a proper selection of the solution methods depending on the spatial discretization scheme used and well location in the area of calculation.

If the radial grid with distributed nodes is used and well is located at the center of the calculational area (for instance, circular), the discrete equation (5) in cylindrical coordinates r, z can be represented as the following balance relation for the block containing the well

$$Q_k = q_{kr+1/2} + q_{k+1/2} - q_{k-1/2} + q_k , \qquad (16)$$

where  $Q_k$  is mass fluid discharge from the block located in the layer "k";  $q_{k\,r\,+\,1/2}$  is mass fluid flux in radial flow across the circumference located at the middle between the first and second nodes of the grid;  $q_{k+1/2}$  is the same for the **ascending** flow of fluid **across** the bottom of the block;  $q_{k-1/2}$  is the same for the fluid flow across the upper cross section of the block;  $q_k$  is mass of fluid extracted from the block volume due to compressibility of the productive zone and vaporization of water.

Discrete equation of fluid flow in the wellbore regarding to the relationship (13) can be represented as follows

$$P_{k+1}^{w} = P_{k}^{w} + \rho_{k+1/2} g(z_{k+1} - z_{k}) + F_{k+1/2} (z_{k+1} - z_{k}) Q_{k+1/2}^{2}, (17)$$

where  $Q_{k+1/2}$  is mass fluid flow rate across the wellbore between the nodes  $z_k$  and  $z_{k+1}$ ;  $\rho_{k+1/2}$ ,  $F_{k+1/2}$  are average values of fluid density and of the coefficient F characterizing the hydraulic resistance in the same interval of the wellbore. The value of  $Q_{k+1/2}$ in the cased interval of the wellbore is to be taken equal to the total mass production of fluid  $\mathcal{Q}_T$  , while in the non-cased (e. g. productive) interval consisting in general of several grid blocks this value must be determined from the following equation of mass balance for the interval of the wellbore between the adjacent grid

$$Q_{k} - Q_{k+1/2} - Q_{k-1/2} = 0 , (18)$$

where  $Q_{k-1/2}$  is mass flow rate in the wellbore between the nodes  $z_k$ and  $z_{k-1}$ .

The compatibility condition for the well and productive zone performance will be satisfied in the considered case if the pressure in a grid block containing the well is equal to the pressure in the wellbore, e. g.

$$p_k = p_k^{\mathbf{w}} \tag{19}.$$

Considering the equations (16) • (18) it is easy to note that it is impossible to find a general solution which will satisfy the condition (19) using any direct method because of non-linear dependence of mass flow rates  $Q_{k+1/2}$  and  $Q_{k-1/2}$  on the pressure drop in the wellbore  $\Delta p_{k+1/2}^{\mathbf{w}}$  and  $\Delta p_{k-1/2}^{\mathbf{w}}$ . It means that the model proposed by K. Aziz and A. Settari (Aziz and Settari, 1982) for the well and productive zone compatibility can not be used in the case of steam and water producing wells and the solution of the equations (16) - (18) has to be found by the iteration procedure.

In the case of well displacement from the center of the calculational area as well as when the block-centered grid is used for spatial discretization the compatibility condition (19) becomes incorrect. In these cases the pressure in a grid block  $p_{lr}$  must satisfy to the mass balance equation similar to the equation (16)

$$Q_{k} = q_{kr+1/2} - q_{kr-1/2} + q_{k+1/2} - q_{k-1/2} + q_{k}, \qquad (20)$$

which differs from (16) only by presence of radial mass flow flux  $q_{kr-1/2}$  from inner contour of the grid block to the center of the calculational area. The pressure  $p_{kr}$  can be considered as the pressure at the circular gallery with the radius  $r_e$  that is equal to the well displacement from the center of the calculational area It is clear that the pressure in the wellbore  $p_k^w$  must be less than the pressure in the grid block  $p_{kr}$  in order to provide fluid extraction from the productive mne. Approximate value of the pressure  $p_k^w$  can be found at the assumption that the fluid discharge from a grid block  $Q_k$  corresponding to the equation (20) is equal to the mass production rate of a well located at the center of a circular reservoir which side surface area is equal to that of the grid block and regards to the circumference with the radius  $r_e$ , where the pressure is equal to the pressure in a grid block  $p_{kr}$  (Fig. 1). Taking into consideration the

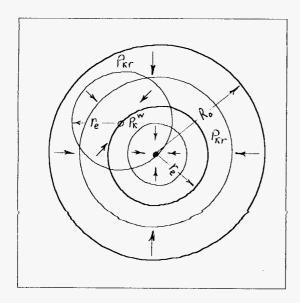


Fig. 1 Schematization of flow to excentrically located well.

possible change of pressure and vapor saturation in a grid block in time fluid discharge rate from the fictitious reservoir can be expressed as an implicit function of these parameters (Aziz and Settari, 1982),

 $Q_k^{n+1} = C_k(p_{kr}^{n+1} - p_k^w) + (C_k)_s(p_{kr}^n - p_k^w)(S_k^{n+1} - S_k^n)_*(21)$  where  $C_k$  is productivity factor of a well producing in the fictitious reservoir;  $(C_k')_s = \partial C_k / \partial S_k$ ;  $p_{kr}^n$ ,  $S_k^n$ ,  $p_{kr}^{n+1}$ ,  $S_k^{n+1}$  -values of pressure and vapor saturation at the time steps n and n+1 respectively.

The solution of the equations (17), (18), (20) and (21) also requires the use of iteration methods. The solution of energy equation (6) for the fluid flow in the productive zone and that for the flow in

the wellbore (12) has to be done after obtaining the pressure values in grid blocks  $p_{k,r}$  and in the wellbore  $p_k^w$  as well as the mass fluid rate distribution  $Q_k$  and  $Q_{k+1/2}$  and  $Q_{k-1/2}$ 

#### 3 RESULTS

To **probe** the developed model it **was** applied to the **interpretation** of field test data obtained **from the** wells **K-8** and **E-1** drilled at the northern part of **Pauzhetka** geothermal field .**The** wells intercepted the **psephite tuff** aquifer with superheated **water** at the depth from **40-90** to **270-350** m (Sugrobov and Kraevoy, **1966**).

The most simple scheme of the well and productive zone interaction was used for the calculations: radial steady state one-phase flow in a homogeneous bed to the well having ideal conditions for fluid entry. That made numerical simulation of the filtration problems in the productive zone unnecessary.

Initial pressure in the **productive zone was** assumed to be equal to hydrostatic one with regard to the depth of its **bedding while** the pressure in the vicinity of the **well was** supposed to be equal to the pressure at the wellbore that has been found from the solution of equation (18) taking into **account** the **values** of the mass production rate  $Q_T$  and well head pressure PI, **accepted**. As the results the pressure at the well bottom (e. g. pressure drop relative to the initial value), the depth of **vaporizing** front in the wellbore and regime of fluid flow in it (**pellet** or annular) were determined for each combination of  $Q_T$  and PI,

Since under considered conditions the dependence of  $\mathcal{Q}_T$  on  $p_h$  is determined by fluid transmissibility kb and its temperature  $t_f$  these two parameters were subjected to stochastic variation within some range of their accepted values made by Monte-Carlo method that was accompanied by numerical simulation of the fluid flow. Results of simulationare presented in Fig. 2.

Testing of the developed numerical program for the simulation of two-phase non-isothermal filtration in the productive zone was done by comparison the results obtained with known numerical and quasi-analytical solutions of the problems concerned with the radial flow in a bomogeneous formation at the different values of initial vapor saturation and constant rate of fluid discharge per 1 m of effective thickness of the reservoir (Fig. 3). As it seen from these data the results of all calculations are in a very good agreement that proves the adequateness of the developed model.

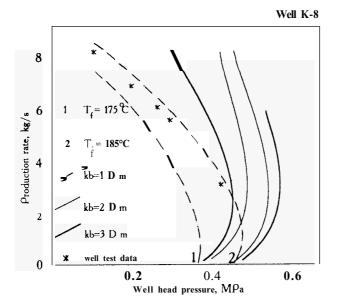
# 4. DISCUSSION OF THE RESULTS

Since the  $Q-p_k$  diagram obtained as the result of numerical simulation (Fig. 2) is influenced by two parameters characterizing the fluid flow in the productive zone (e. g. fluid transmissibility kb and its temperature  $t_f$ ) they can be considered as the additional conditions of the well and productive zone compatibility besides the condition (19) prescribed before. Comparing the field well test data with the results of numerical simulation of the well and productive zone compatible performance combined with the stochastic simulation of their compatibility conditions makes it possible to estimate the value of these parameters.

To get a definite result of such estimation it is important to run the test at each step of the well head pressure long enough to provide quasi-stationary fluid flow in the productive zone and heat exchange in the wellhore.

## 5. CONCLUSIONS

The results of the fulfilled studies show that numerical simulation of the well and productive zone compatible performance combined with stochastic simulation of their compatibility conditions is capable to serve as an effective mean to increase the informativeness of all testing works at high temperature hydrothermal reservoirs.



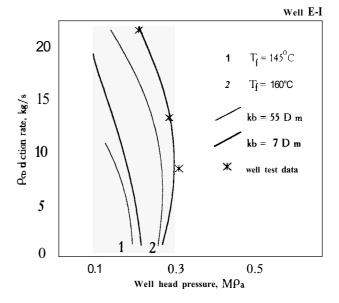
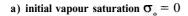
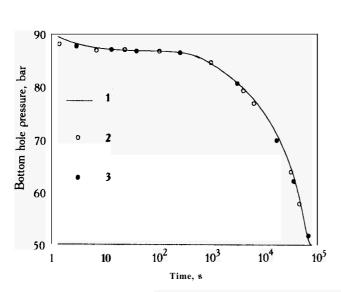
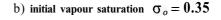


Fig. 2. Calculated production rate vs well head pressure.







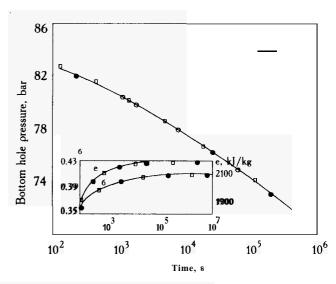


Fig. 3. Comparison of the simulation results (3) with other solutions of the test problem: by TOUGH Program (1a), Garg (2), O'Sullivan (4), Grant (1b) (Grant and Sorry, 1979).

	<b>CHOMENCIATURE</b>
a, C	-coefficients in finite difference equations
$a_{\bullet}$	-thermal conductivity
b	-effective thickness of the productive zone
Bi	-Biot criterion
$c_0$	-specific heat capacity
C,	-well productivity factor
e	-enthalpy of the fluid
$\boldsymbol{E}$	-specific volume enthalpy of the productive zone
Fo	-Fourier number
g	-gravity acceleration
h	-spatial step size

i, j	-grid node number
k	-absolute permeability
k,	-heat transfer coefficient
K,	-transient heat exchange coefficient
m	-porosity
n	-time-step number
p	-pressure
q	-mass fluid flux
$q_{m_i}, q_s$	-sources of mass and energy
$q_w$	-heat exchange intensity between the fluid flow
	in the wellbore and surrounding rock
Q	-mass fluid flow rate

r	-radial coordinate
$r_{i,j}$	-grid function of mass fluid flux
R', R"	-relative liquid and vapor phase permeabilities
S	-vapor saturation of the fluid
t	-temperature
T	-fluid mobility coefficients
T',T"	-fluid phase mobility coefficients
v	-fluid flow flux
V	-mass fluid flow flux
V ',V "	-liquid and vapor phase mass fluxes
x, z	-horisontal and vertical coordinates
$\alpha_0$	-heat exchange coefficient
$\boldsymbol{\beta}_{\star}$	-rock compressibility
<i>8'</i>	-fluid and rock matrix compressibility
$\beta_{i}$ $\delta'_{c}$ , $\delta_{i}$	-thickness of the cement and casing pipe
a	-productive zone heat conductivity coefficient
λ',λ"	-liquid and vapor phase heat conductivity
	coefficients
$\mu'$ , $\mu$ "	-fluid phase dynamic viscosities
η	-mass flux vapor saturation of the fluid
ρ	-fluid density
$\rho', \rho''$	-liquid and vapor phase densities
$\rho_0$	-rock density
τ	-time
$\tau_k$	-shear stress at the flow - wellbore wall contact
$\varphi_1, \varphi_2$	-source terms in finite difference equations
Ψ	-real vapor saturation of the fluid flow in the wellbore

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